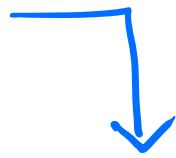


10.10.2022

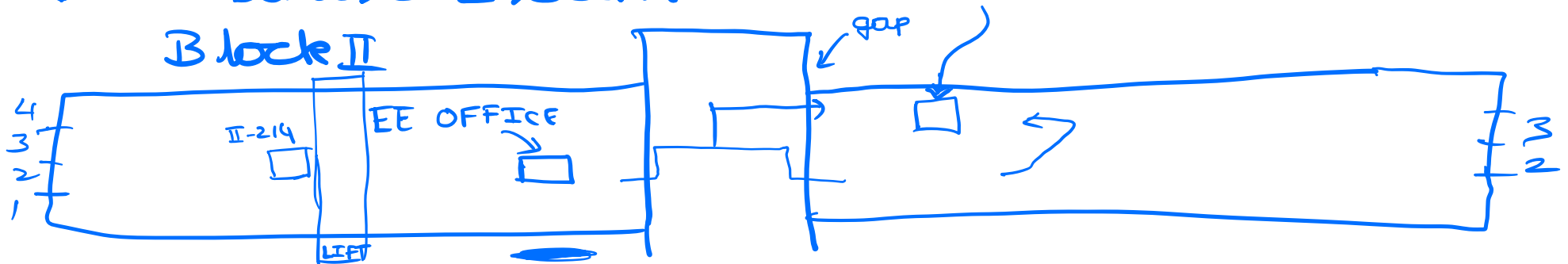
- Check calculation of 1D example design



[https://web.iitd.ac.in/~shaunak/sen/2022Sem1\\_ELL333/ELL333\\_Design\\_1Dexample.jl.html](https://web.iitd.ac.in/~shaunak/sen/2022Sem1_ELL333/ELL333_Design_1Dexample.jl.html)

- Project report → submit in next class (LH 316)

- Minor Exam → II-339 B



①

$$\dot{x} = Ax + Bu$$

$n \times 1$        $m \times 1$

$$y = Cx + Du$$

$p \times 1$

• Transfer Function

Laplace Transform of

$$sX(s) - x(0) = AX(s) + BU(s)$$

$\uparrow = 0$

$$(sI - A)X(s) = BU(s)$$

Transfer Function  $\rightarrow$

$$X(s) = (sI - A)^{-1} BU(s)$$

$$Y(s) = (C(sI - A)^{-1} B + D) U(s)$$

# ⊙ Transfer Function

$$C(sI - A)^{-1}B + D$$

• If  $(A, B, C, D) \xrightarrow{\text{similarity}} (TAT^{-1}, TB, CT^{-1}, D)$ ,

then transfer function is ?

$$CT^{-1} (sI - \underset{\substack{\uparrow \\ TT^{-1}}}{TAT^{-1}})^{-1} TB + D$$

$$= CT^{-1} (T(sI - A)T^{-1})^{-1} TB + D$$

$$= \underbrace{CT^{-1}(T^{-1})^{-1}}_I (sI - A)^{-1} \underbrace{T^{-1}TB}_I + D$$

= same, as the relationship between  $u$  &  $y$  unchanged.

$$\begin{aligned} z &= Tx \\ \dot{z} &= T\dot{x} \\ &= TA x + TBu \\ &= TAT^{-1}z + TBu \\ y &= CT^{-1}z + Du \end{aligned}$$

$$\textcircled{\circ} \text{ Transfer Function} = \frac{N(s)}{D(s)} \quad \leftarrow \text{matrix}$$

$$(sI - A)^{-1} = \frac{\text{adj} \{ (sI - A) \}}{\det \{ (sI - A) \}} \quad \leftarrow$$

Roots of  $D(s) = \det \{ (sI - A) \}$   
are poles  $\equiv$  eigenvalues of  $A$ .

Calculate transfer function

$$A = \begin{bmatrix} -1 & 0 & 0 & 0 \\ 0 & -2 & 0 & 0 \\ 0 & 0 & -3 & 0 \\ 0 & 0 & 0 & -4 \end{bmatrix}_{4 \times 4}, \quad B = \begin{bmatrix} 1 \\ 0 \\ -1 \\ 0 \end{bmatrix}_{4 \times 1}$$

$$C = [1 \ 1 \ 0 \ 0]_{1 \times 4}, \quad D = 0$$

$$C(sI - A)^{-1}B = \frac{1}{(s+2)(s+3)(s+4)}$$

$\frac{1}{s+1}$  ← ?

$$\frac{(s+2)(s+3)(s+4)}{(s+1)(s+2)(s+3)(s+4)}$$

Pole-Zero  
cancellation

13.10.2022

$$\begin{array}{c}
 C \quad (sI - A)^{-1} \quad B = \frac{1}{(s+2)(s+3)(s+4)} \\
 \begin{array}{ccc}
 1 \times 4 & 4 \times 4 & 4 \times 1
 \end{array}
 \end{array}$$

$\checkmark \frac{1}{s+1}$

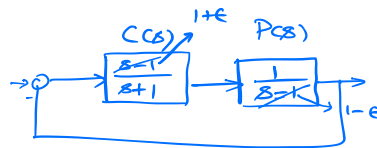
$$\frac{(s+2)(s+3)(s+4)}{(s+1)(s+2)(s+3)(s+4)}$$

Arrows indicate the cancellation of  $(s+2)$ ,  $(s+3)$ , and  $(s+4)$  in the numerator and denominator. A question mark is placed above the cancelled terms in the denominator.

Pole-Zero  
cancellation

## → Pole-Zero Cancellation

- indicates loss of controllability/observability



- practically not possible  $\Rightarrow$  responses may "blow up".

$$Y(s) = G(s) U(s) \quad m \times 1$$

p x 1

$$G(s) = C(sI - A)^{-1} B + D$$

p x m

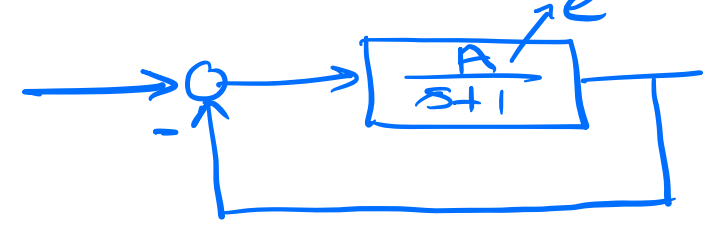
$$\left[ \begin{array}{cccc} G_{11}(s) & G_{12}(s) & \dots & G_{1m}(s) \\ G_{21}(s) & G_{22}(s) & \dots & G_{2m}(s) \\ \dots & \dots & \dots & \dots \\ G_{p1}(s) & G_{p2}(s) & \dots & G_{pm}(s) \end{array} \right] \quad p \times m$$

p m transfer functions need to be visualised.

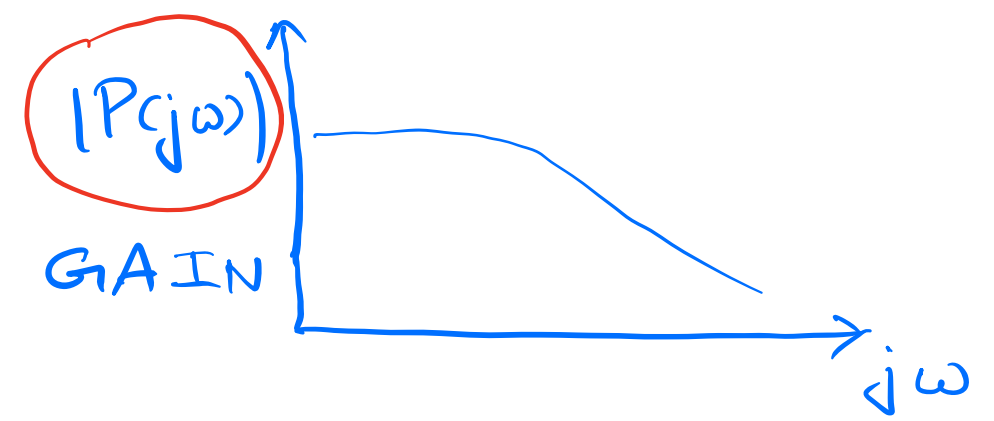
What do we associate with a transfer function in design of Single Input Single Output (SISO) designs (ELL225)?

Example:  $P(s) = \frac{A}{s+1} \cdot e^{-Ts}$

... Bode plots  
Nyquist plots



$$s + 1 + A e^{-Ts} = 0$$





# Gain for multivariable

system's frequency response

- gain of  $G(s)$ ?  
 $p \times m$
- gain/size of a vector  $\begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix}$ ? = norms
- gain/size of a matrix  $A$  (need not be square)? = induced norm (singular values)

gain / size of a vector  $\begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix}$ ?

= norm / distance

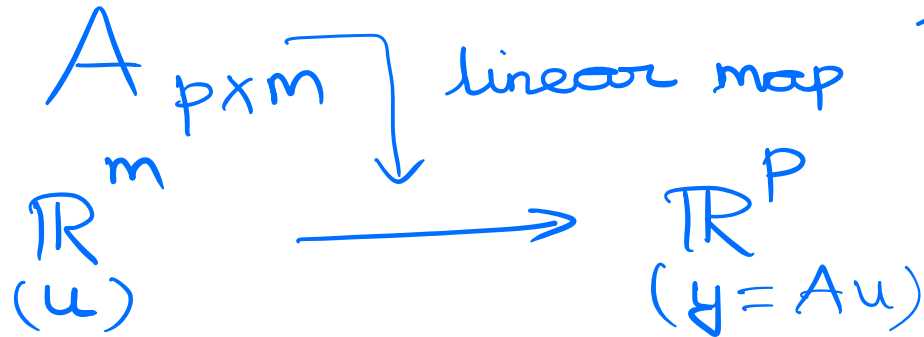
✓  $\hookrightarrow \sqrt{x_1^2 + x_2^2 + \dots + x_n^2}$  Euclidean  
2-norm

$\rightarrow (|x_1| + |x_2| + \dots + |x_n|)$  1-norm

$\rightarrow (|x_1|^p + |x_2|^p + \dots + |x_n|^p)^{1/p}$  p-norm

✓  $\rightarrow \max\{|x_1|, |x_2|, \dots, |x_n|\}$   $\infty$ -norm

gain / size of a matrix  $A$   
(need not be square)



= induced matrix norm, norm

induced by the vector norms

$$\|A\|_2 = \max_u \frac{\|y\|_2}{\|u\|_2} \quad (\text{Is this a norm?})$$

as  $\|u\|_2^2 = u'u$

$$\|y\|_2^2 = y'y = (Au)'(Au) = u'A'Au$$

$$\Rightarrow \|A\|_2 = \max_u \sqrt{\frac{u'A'Au}{u'u}}$$

$$u' \underbrace{A'A}_{M} u$$

$M \rightarrow$  symmetric matrix  
 $m \times m$  square

eigenvalues:  $\lambda_1, \lambda_2, \dots, \lambda_m$  denote the maximum by  $\lambda_{\max}$

[ $\triangleq$  singular values of  $A$ ]

In the eigenvector direction,  $\bar{u}$

corresponding to  $\lambda_{\max}$ ,

$$\|A\|_2 = \max_{\substack{u \\ = \bar{u}}} \sqrt{\frac{\bar{u}' M \bar{u}}{\bar{u}' \bar{u}}} = \sqrt{\frac{\bar{u}' \lambda_{\max} \bar{u}}{\bar{u}' \bar{u}}}$$

$$= \sqrt{\lambda_{\max}} \sqrt{\frac{\bar{u}' \bar{u}}{\bar{u}' \bar{u}}} = \sqrt{\lambda_{\max}}$$

Induced matrix norm  $\|A\|_2 = \sqrt{\lambda_{\max}(A'A)}$   
is this positive

$$\begin{aligned} \rightarrow u'A'Au &= (uA)'(Au) \\ &= \|Au\|_2^2 \\ &\geq 0 \end{aligned}$$

$u' \underbrace{A'A}_M u$  : Quadratic Form

$v = \begin{bmatrix} x \\ y \end{bmatrix} \in \mathbb{R}^{2 \times 1}$

$v'v = x^2 + y^2$

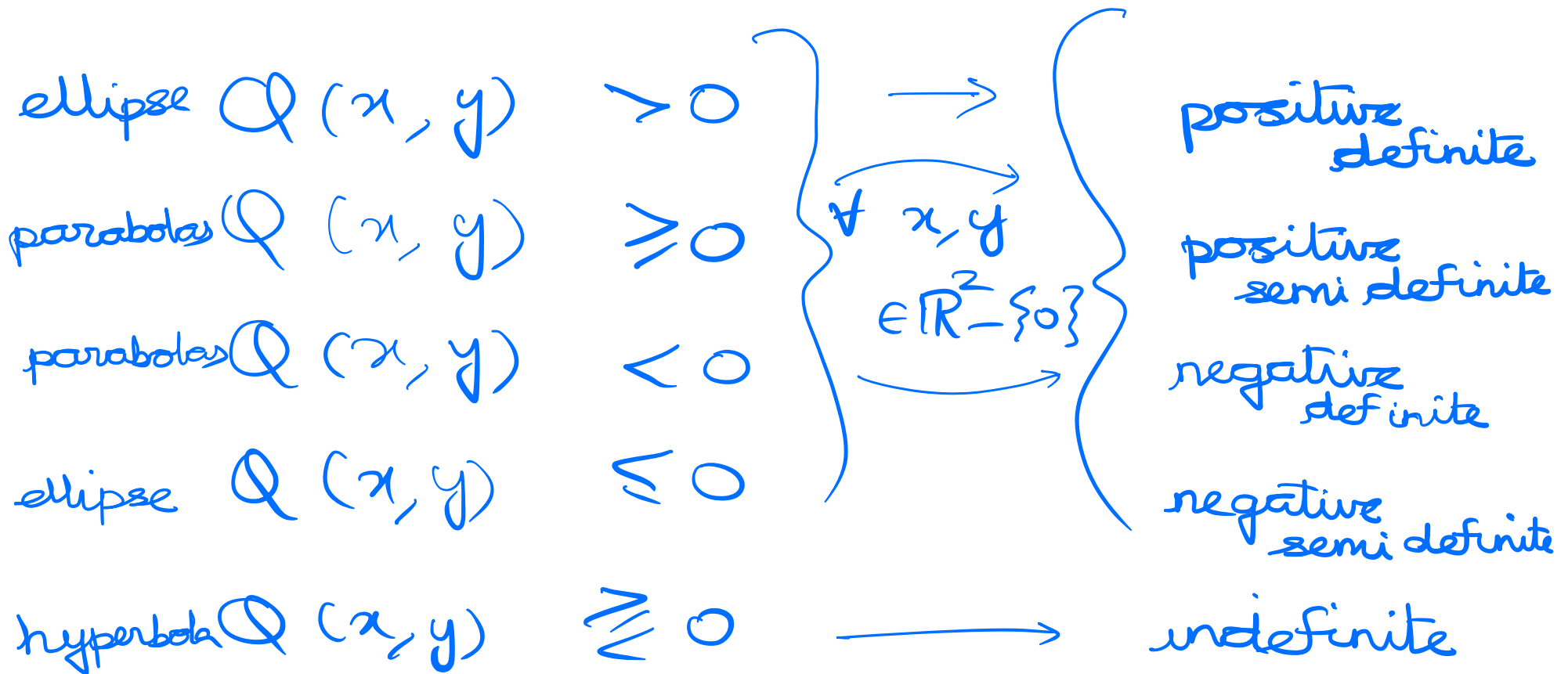
$v' M v = \begin{bmatrix} x & y \end{bmatrix} \begin{bmatrix} a & b/2 \\ b/2 & c \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix}$   
↓  
symmetric

$\begin{bmatrix} a & b/2 \\ b/2 & c \end{bmatrix} = \begin{bmatrix} ax + by & \frac{bx}{2} + cy \\ \frac{bx}{2} + cy & \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix}$

$Q(x, y) = ax^2 + bxy + cy^2$

For a fixed  $M$  on  $\{a, b, c\}$ ,

is the sign of  $Q(x, y)$  definitely known?



17.10.2022

• Transfer Function  
matrix  $\rightarrow$  Gain?

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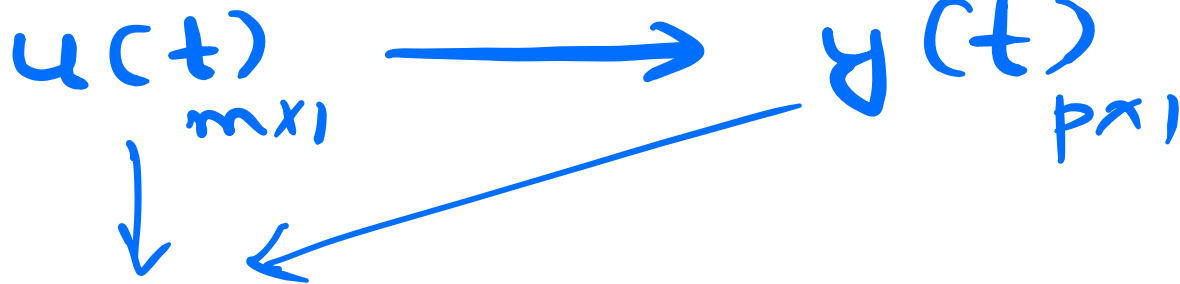
① static  $v = Av$

- "size" of a vector  
~ norm, ex:  $\|\cdot\|_2$
- "size" of a matrix  
~ induced norm  
 $\Rightarrow \|A\|_2 \triangleq \max_x \frac{\|Ax\|_2}{\|x\|_2}$



dynamic map

$$\bullet \quad y(t) = \int_0^t e^{A(t-z)} B u(z) dz$$



Vector space not  $\mathbb{R}^n$

but  $L_2[0, \infty) = \{ u(t) : \int_0^\infty |u(t)|^2 dt < \infty \}$

$$L_2 \text{ norm}(u) = \int_0^\infty |u(t)|^2 dt$$

induced norms  
have to be  
defined in  
this space...

- $\int_0^{\infty} |y(t)|^2 dt = ( ) \int_0^{\infty} |Y(j\omega)|^2 d\omega$

$$= ( ) \int_0^{\infty} |G(j\omega) U(j\omega)|^2 d\omega$$

$$= ( ) \int_0^{\infty} |G(j\omega)|^2 |U(j\omega)|^2 d\omega$$

If  $G$  was  
scalar  
frequency  
response

$$\leq \max_{\omega} |G(j\omega)|^2 ( ) \int_0^{\infty} |U(j\omega)|^2 d\omega$$

$$\leq \max_{\omega} |G(j\omega)|^2 \int_0^{\infty} |u(t)|^2 dt$$

If  $G$  was vector  $\rightarrow \|G\|_{\infty}$

→ Robust Control

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→ Optimal Control

→ Stochastic Filtering & Identification

→ ~ Robotics

→ ~ Adaptive Control

→ Special Topics

# Discrete Models

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx + Du\end{aligned}$$

$x(t)$   
 $t$ : continuous  
above below  
 $A \neq A$

↓ same notation, but different meaning

$$x_{k+1} = Ax_k + Bu_k \quad x[k]$$

$$y_k = Cx_k + Du_k \quad \equiv x_k$$

Why discrete? ↗ ease of computing  
↳  $\Sigma$  instead of  $\int$

# Stability

$$\dot{x} = Ax, \text{ eig}(A) \in \text{LHP} \Rightarrow \text{Stability}$$

$$x_{k+1} = A x_k$$

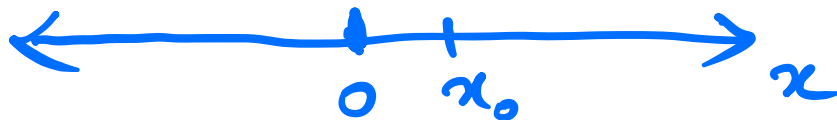
$$= A \cdot A x_{k-1}$$

$$= A \cdot A \dots A \cdot x_0$$

← (k+1) times

$$= A^{k+1} x_0$$

Stability at  $x=0$



What is condition on  $A$  so that  $x_k \rightarrow 0$ ?