

Enthalpy of Moist Air

- Gibb's law states:

$$h_m = h_a + wh_{wv}$$

$$h_a = C_{pa} \cdot t \quad (\text{assuming datum at } 0 \text{ } ^\circ\text{C})$$

h_{wv} = superheated steam at p_{wv} & t

It can be approximated as (at low pressure)

$$h_{wv} = 2501 + C_{pv} \cdot t$$

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 $h_{fg} (0^\circ\text{C})$ 1.86 kJ/kgK

At low pressure for an ideal gas, enthalpy is a function of temperature only.

Humid Specific Heat

$$h = (C_{pa} + wC_{pv})t + wh_{fg}(0^{\circ}C)$$
$$= C_{pm}t + wh_{fg}(0^{\circ}C)$$

Where $C_{pm} = 1.006 + w 1.86$

An approx. value of c_{pm} is 1.0216 kJ/kgK (Arora, 2009)

Specific volume

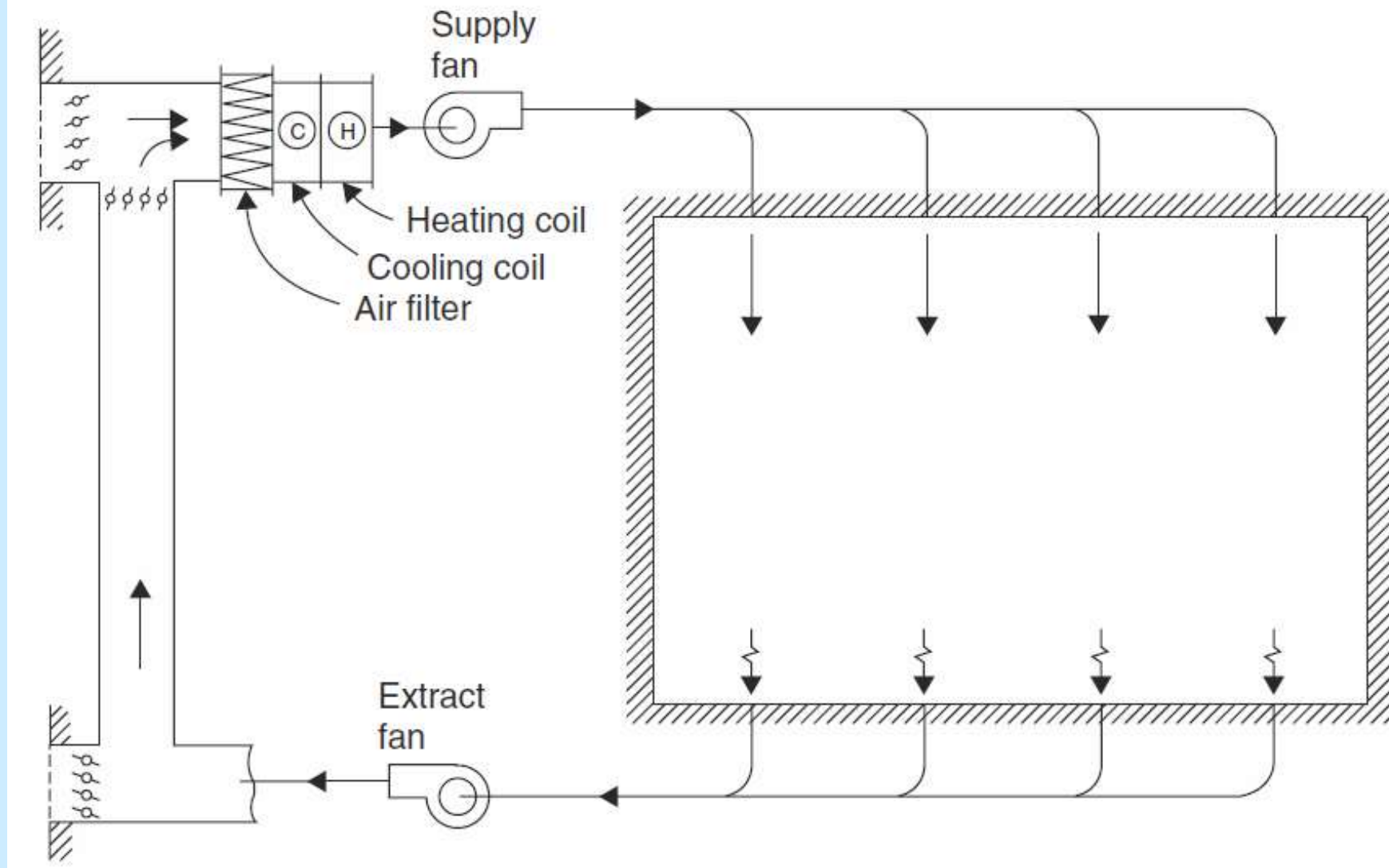
- Volume of moist air per unit mass of dry air

$$v = \frac{RT}{28.966(p - p_w)} = \frac{R_{da}T}{p - p_w}$$

Common Psychrometric Processes

- Sensible heating/ Cooling
- Mixing of air streams
- Cooling and dehumidification
- Steam injection
- Evaporative Cooling
- Chemical dehumidification (e.g. over Silica gel)
- Spray washers with heating/ cooling of water

Central air-handling unit – basic circuit



Input Parameters for design

- Outside and Inside design conditions
- % ventilation or ventilation air flow rate
- Room loads (through calculations)
 - RSHF

Important Concepts in Air-Conditioning

- Coil Condition curve
- Coil ADP
- Bypass factor
- Room sensible heat factor (RSHF)

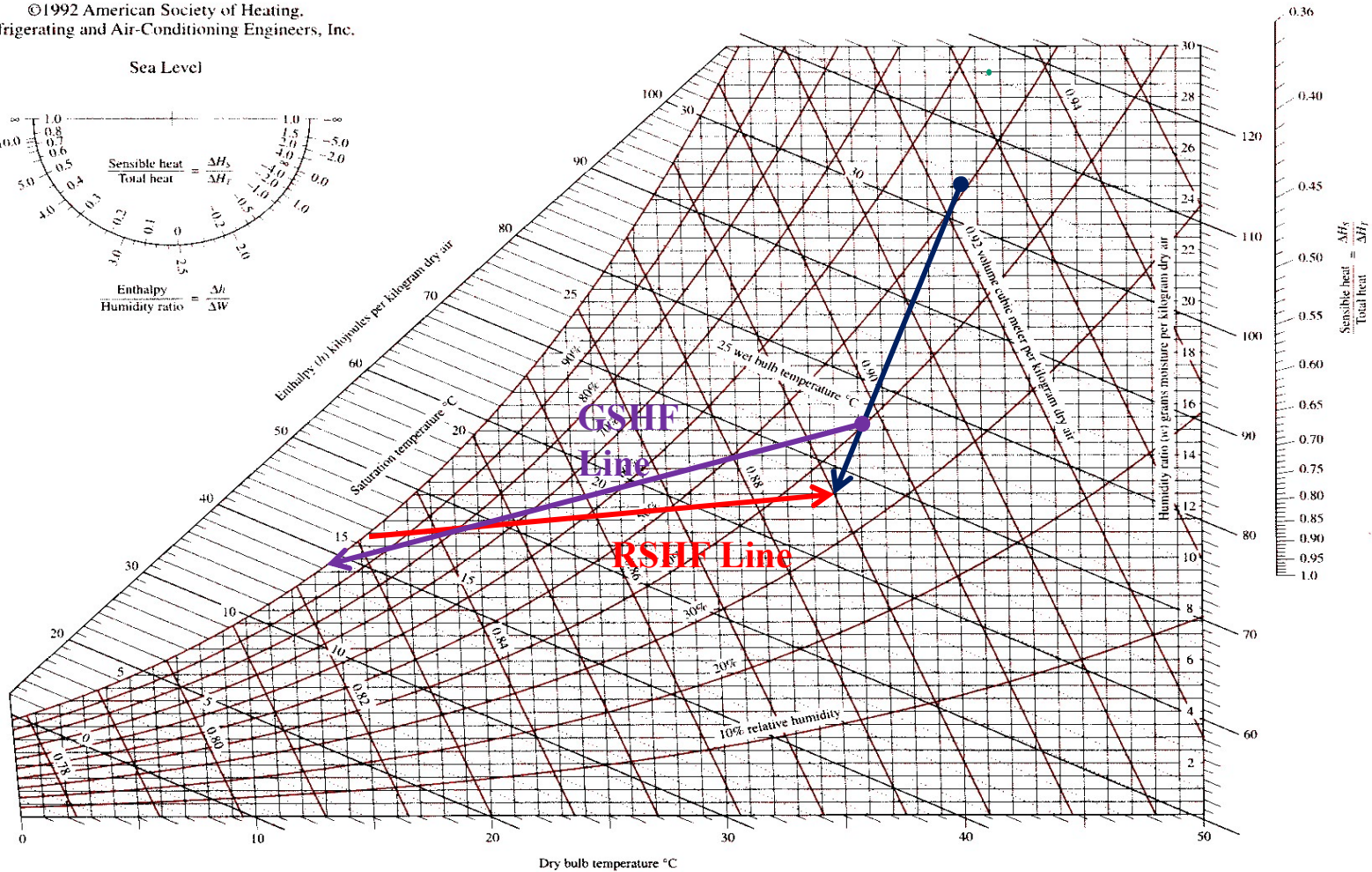
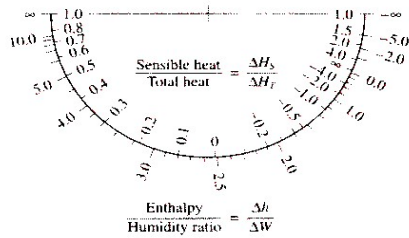
Summer Air-conditioning with Bypass

ASHRAE Psychrometric Chart No. 1
Normal Temperature
Barometric Pressure: 101.325 kPa



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Sea Level



Performance Parameters

- Mass flow rate of supply air
- Room loads (TR)
- Cooling capacity of system
- COP of system